

# ENGINE DESIGN AND OPERATING PARAMETERS

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# IMPORTANT ENGINE CHARACTERISTICS

- In this chapter, some basic geometrical relationships and the parameters commonly used to characterize engine operation are developed.
- The factors important to an engine user are:
  1. The engine's performance over its operating range
  2. The engine's fuel consumption within this operating range and the cost of the required fuel
  3. The engine's noise and air pollutant emissions within this operating range
  4. The initial cost of the engine and its installation.

5. The reliability and durability of the engine, its maintenance requirements, and how these affect engine availability and operating costs

- These factors control total engine operating costs-usually the primary consideration of the user-and whether the engine in operation can satisfy environmental regulations.
- This course is concerned primarily with the performance, efficiency, and emissions characteristics of engines; the omission of the other factors listed above does not, in any way, reduce their great importance.

Engine performance is more precisely defined by:

1. The maximum power (or the maximum torque) available at each speed within the useful engine operating range
2. The range of speed and power over which engine operation is satisfactory

The following performance definitions are commonly used:

- ❑ **Maximum rated power.** The highest power an engine is allowed to develop for short periods of operation.
- ❑ **Normal rated power.** The highest power an engine is allowed to develop in continuous operation.
- ❑ **Rated speed.** The crankshaft rotational speed at which rated power is developed.

# GEOMETRICAL PROPERTIES OF RECIPROCATING ENGINES

The following parameters define the basic geometry, of a reciprocating engine (see Fig. 2.1):

**Compression ratio  $r_c$  :**

$$r_c = \frac{\text{maximum cylinder volume}}{\text{minimum cylinder volume}} = \frac{V_d + V_c}{V_c} \quad (2.1)$$

where  $V_d$ , is the displaced or swept volume and  $V_c$  is the clearance volume.

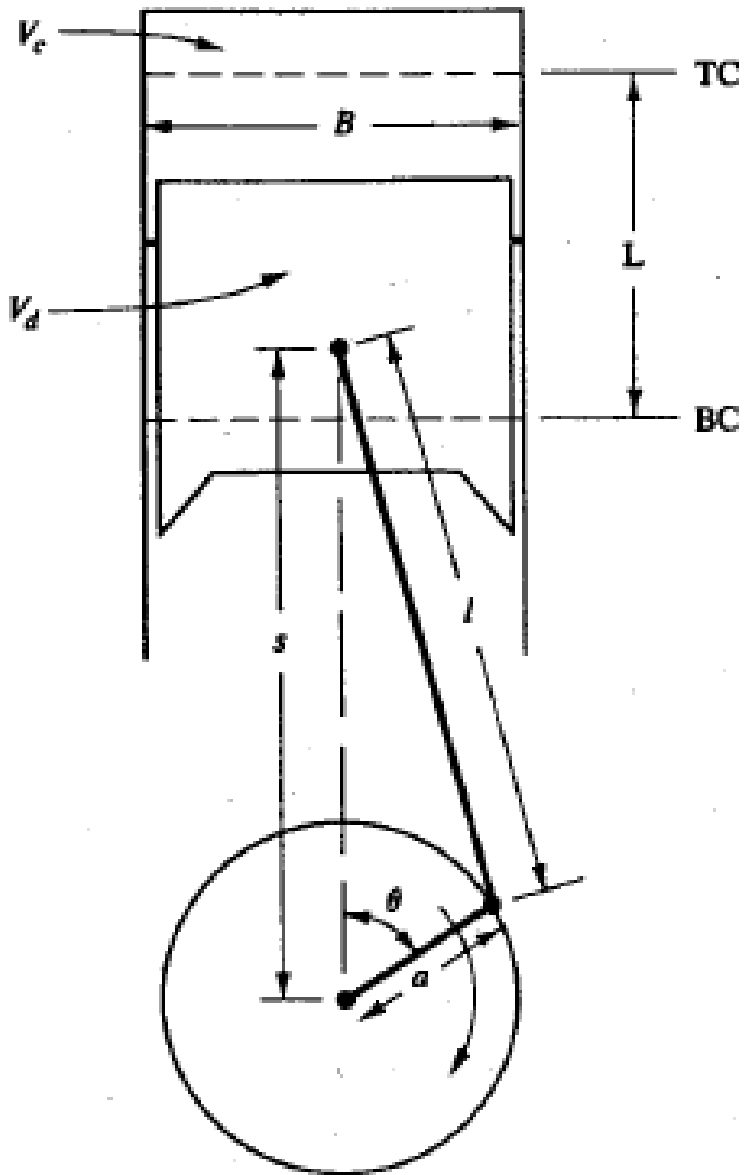


FIGURE 2-1

Geometry of cylinder, piston, connecting rod, and crankshaft where  $B$  = bore,  $L$  = stroke,  $l$  = connecting rod length,  $a$  = crank radius,  $\theta$  = crank angle.

- Ratio of cylinder bore to piston stroke:

$$R_{bs} = \frac{B}{L}$$

- Ratio of connecting rod length to crank radius:

$$R = \frac{l}{a}$$

- The stroke and crank radius are related by

$$L = 2a$$

- Typical values of these parameters are:  $r_c = 8 - 12$  for SI engines and  $12 - 24$  for CI engines;
- $B/L = 0.8$  to  $1.2$  for small- and medium-size engines, decreasing to about  $0.5$  for large slow-speed CI engines;
- $R = 3 - 4$  for small-and medium-size engines, increasing to  $5 - 9$  for large slow-speed CI engines.



The cylinder volume  $V$  at any crank position  $\theta$  is

$$V = V_c + \frac{\pi B^2}{4} (l + a - s)$$

where  $s$  is the distance between the crank axis and the piston pin axis (Fig. 2-1), and is given by

$$s = a \cos \theta + (l^2 - a^2 \sin^2 \theta)^{1/2}$$

The angle  $\theta$ , defined as shown in Fig. 2-1, is called the ***crank angle***.

Equation (2.4) with the above definitions can be rearranged:

$$\frac{V}{V_c} = 1 + \frac{1}{2} (r_c - 1) [R + 1 - \cos \theta - (R^2 - \sin^2 \theta)^{1/2}]$$

The combustion chamber surface area  $A$  at any crank position  $\theta$  is given by

$$A = A_{ch} + A_p + \pi B(l + a - s)$$

where  $A_{ch}$  is the cylinder head surface area and  $A_p$  is the piston crown surface area. For flat-topped pistons,  $A_p = \pi B^2/4$ .

Using Eq. (2.5), Eq. (2-7) can be rearranged

$$A = A_{ch} + A_p + \frac{\pi BL}{2} [R + 1 - \cos \theta - (R^2 - \sin^2 \theta)^{1/2}]$$

An important characteristic speed is the mean piston speed  $S_p$  :

$$\bar{S}_p = 2LN$$

where  $N$  is the rotational speed of the crankshaft.

- Mean piston speed is often a more appropriate parameter than crank rotational speed for correlating engine behavior as a function of speed.
- For example, gas-flow velocities in the intake and the cylinder all scale with  $S_p$ . The instantaneous piston velocity  $S_p$  is obtained from

$$S_p = \frac{ds}{dt}$$

- The piston velocity is zero at the beginning of the stroke, reaches a maximum near the middle of the stroke, and decreases to zero at the end of the stroke.
- Differentiation of Eq. (2.5) and substitution gives

$$\frac{S_p}{\bar{S}_p} = \frac{\pi}{2} \sin \theta \left[ 1 + \frac{\cos \theta}{(R^2 - \sin^2 \theta)^{1/2}} \right]$$

- Figure 2-2 shows how  $S_p$  varies over each stroke for  $R = 3.5$

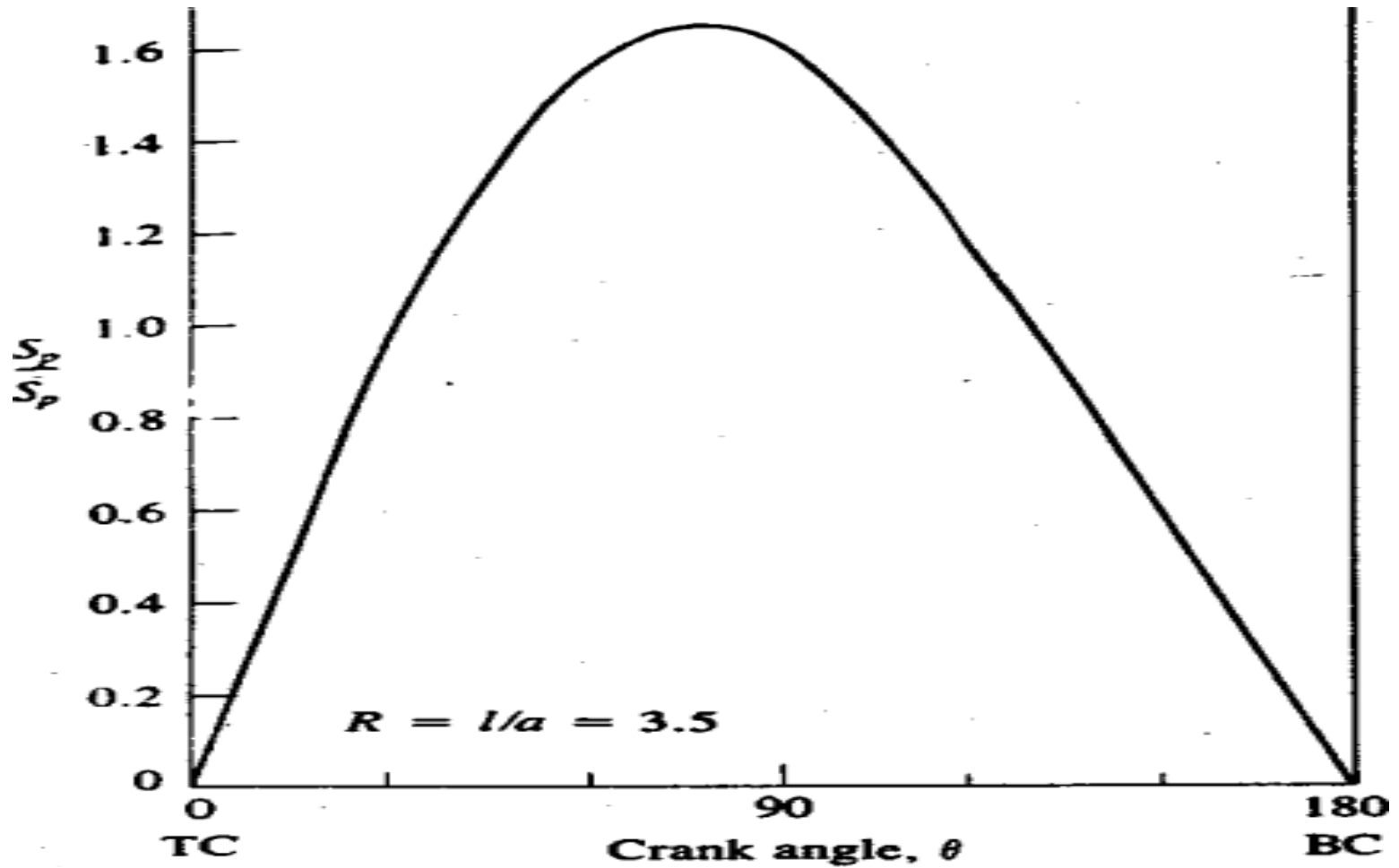


FIGURE 2-2 Instantaneous piston speed/mean piston speed as a function of crank angle for  $R = 3.5$ .

- Resistance to gas flow into the engine or stresses due to the inertia of the moving parts limit the maximum mean piston speed to within the range 8 to 15 m/s (1500 to 3000 ft/min).
- Automobile engines operate at the higher end of this range; the lower end is typical of large marine diesel engines.

## 2.3 BRAKE POWER AND TORQUE

- Engine torque is normally measured with a dynamometer.
- The engine is clamped on a test bed and the shaft is connected to the dynamometer rotor.
- Figure 2-3 illustrates the operating principle of a dynamometer.

The rotor is coupled electromagnetically, hydraulically, or by mechanical friction to a stator, which is supported in low friction bearings.

The stator is balanced with the rotor stationary.

The torque exerted on the stator with the rotor turning is measured by balancing the stator with weights, springs, or pneumatic means.



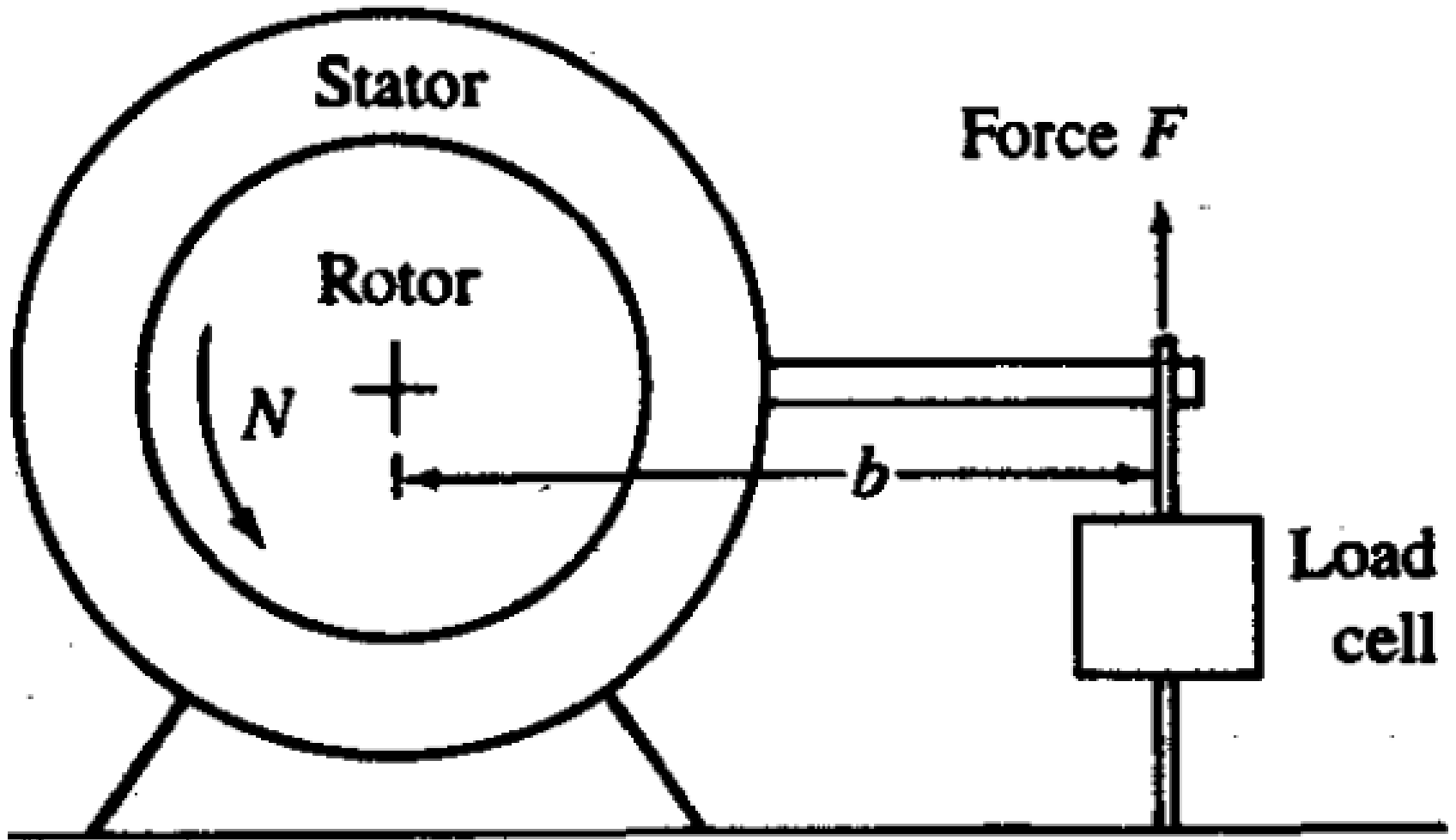
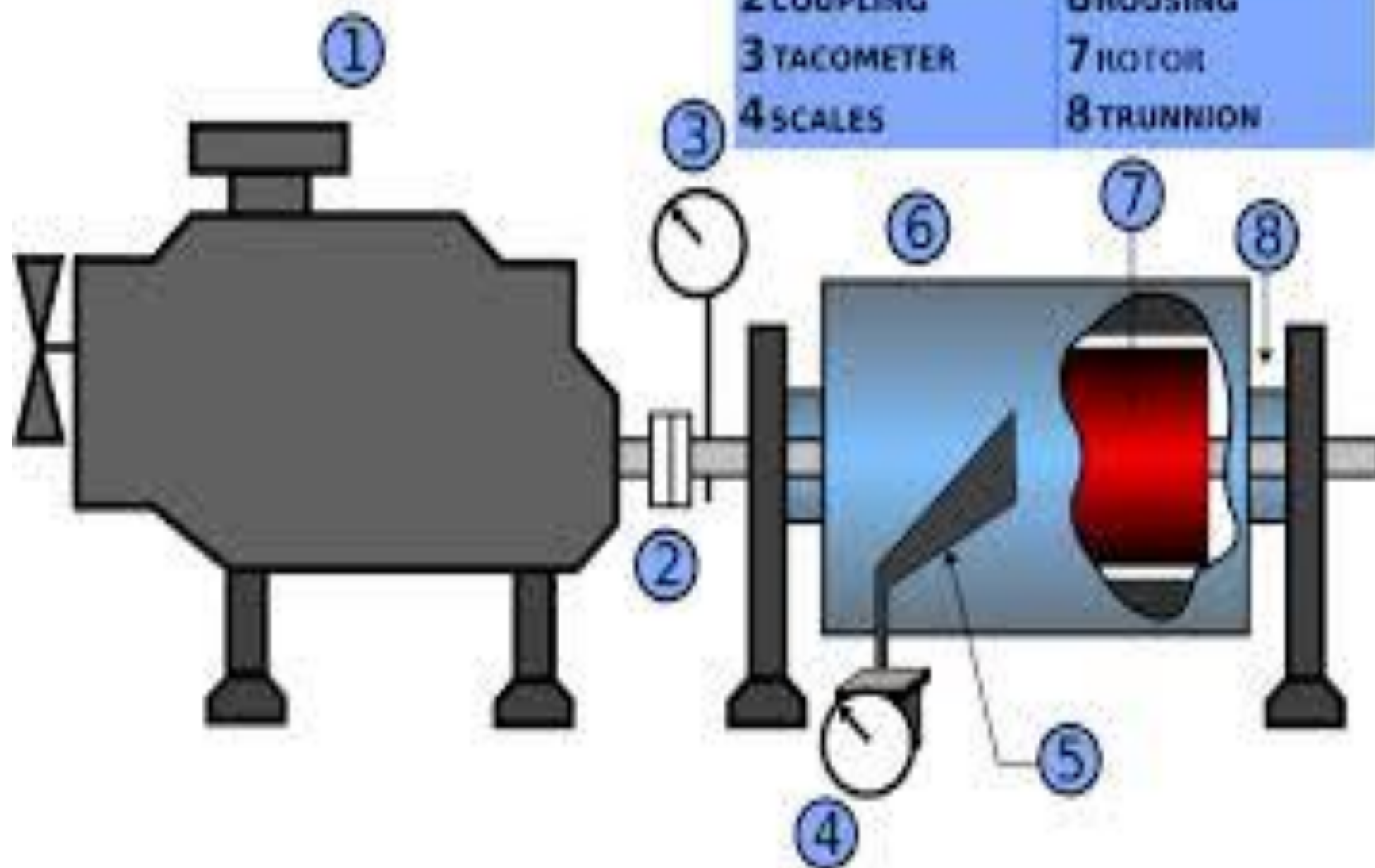
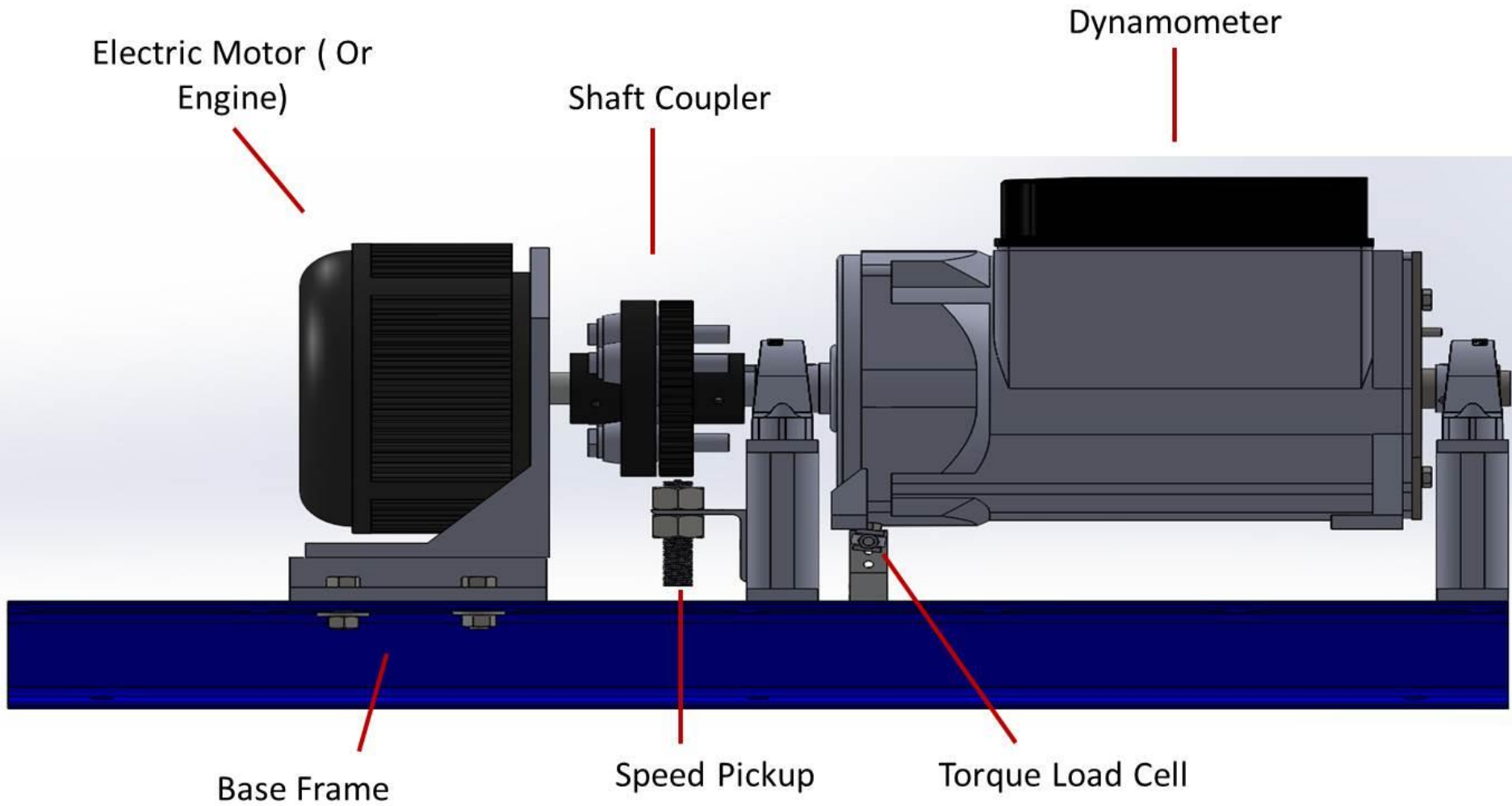


FIGURE 2-3 Schematic of principle of operation of dynamometer

# DYNAMOMETER





Using the notation in Fig. 2-3, if the torque exerted by the engine is  $T$ :

$$T = Fb \quad (2.12)$$

The power  $P$  delivered by the engine and absorbed by the dynamometer is the product of torque and angular speed:

$$P = 2\pi NT \quad (2.13a)$$

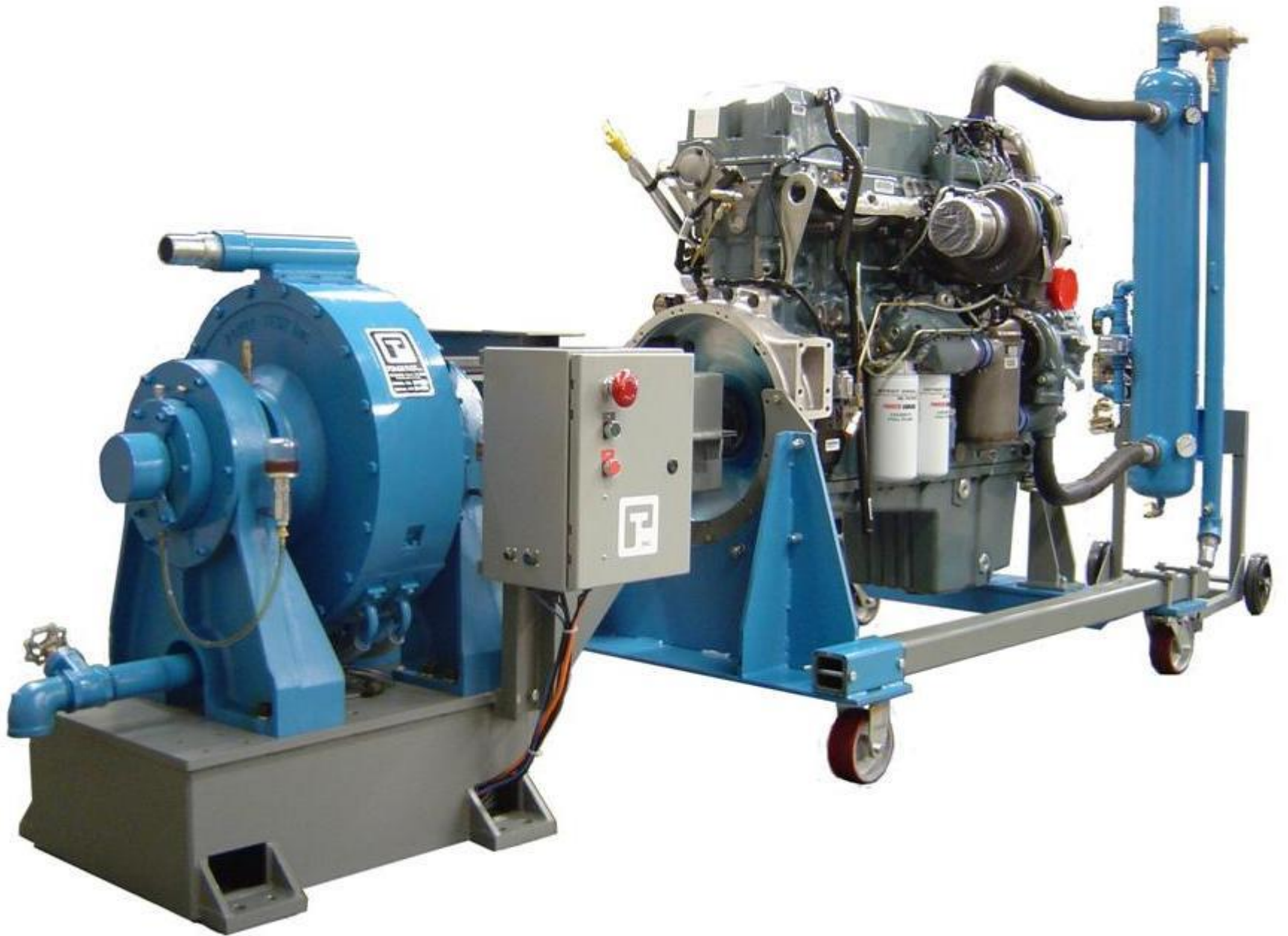
where  $N$  is the crankshaft rotational speed. In SI units:

$$P(\text{kW}) = 2\pi N(\text{rev/s})T(\text{N}\cdot\text{m}) \times 10^{-3} \quad (2.13b)$$

or in U.S. units:

$$P(\text{hp}) = \frac{N(\text{rev/min}) T(\text{lbf}\cdot\text{ft})}{5252} \quad (2.13c)$$

- ❑ Note that torque is a measure of an engine's ability to do work; power is the rate at which work is done.
- ❑ The value of engine power measured as described above is called brake power  $P_b$ .
- ❑ This power is the usable power delivered by the engine to the load-in this case, a "brake."









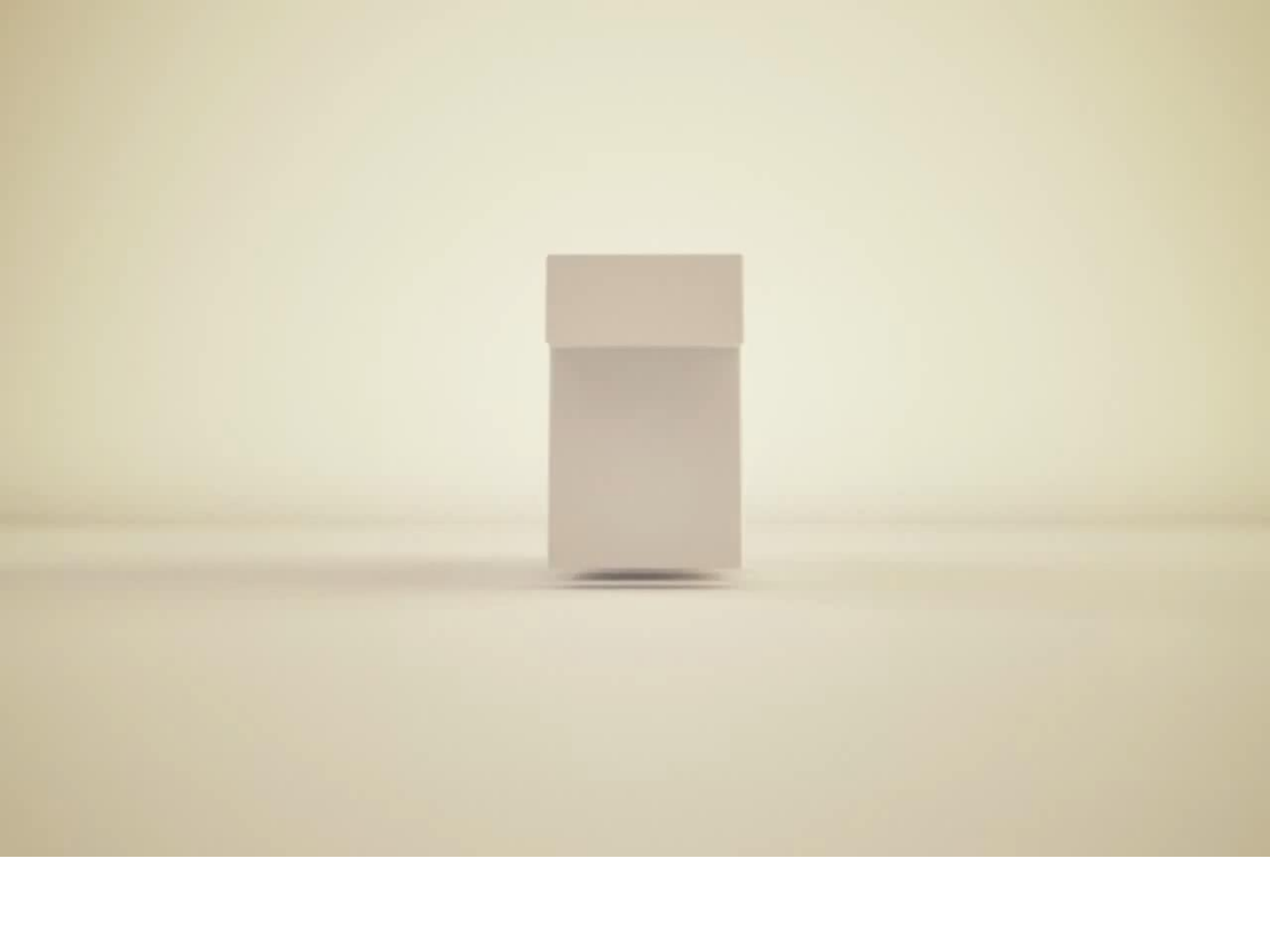








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