

# ENERGY EFFICIENCY ON PUMP SYSTEMS

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# **Pumps and Pumping System**

**2.1 Introduction**

**2.2 Type of pumps**

**2.3 Assessment of pumps**

**2.4 Energy efficiency opportunities**

# Introduction

- **20% of world's electrical energy demand**
- **25-50% of energy usage in some industries**
- **Used for**
  - **Domestic, commercial, industrial and agricultural services**
  - **Municipal water and wastewater services**

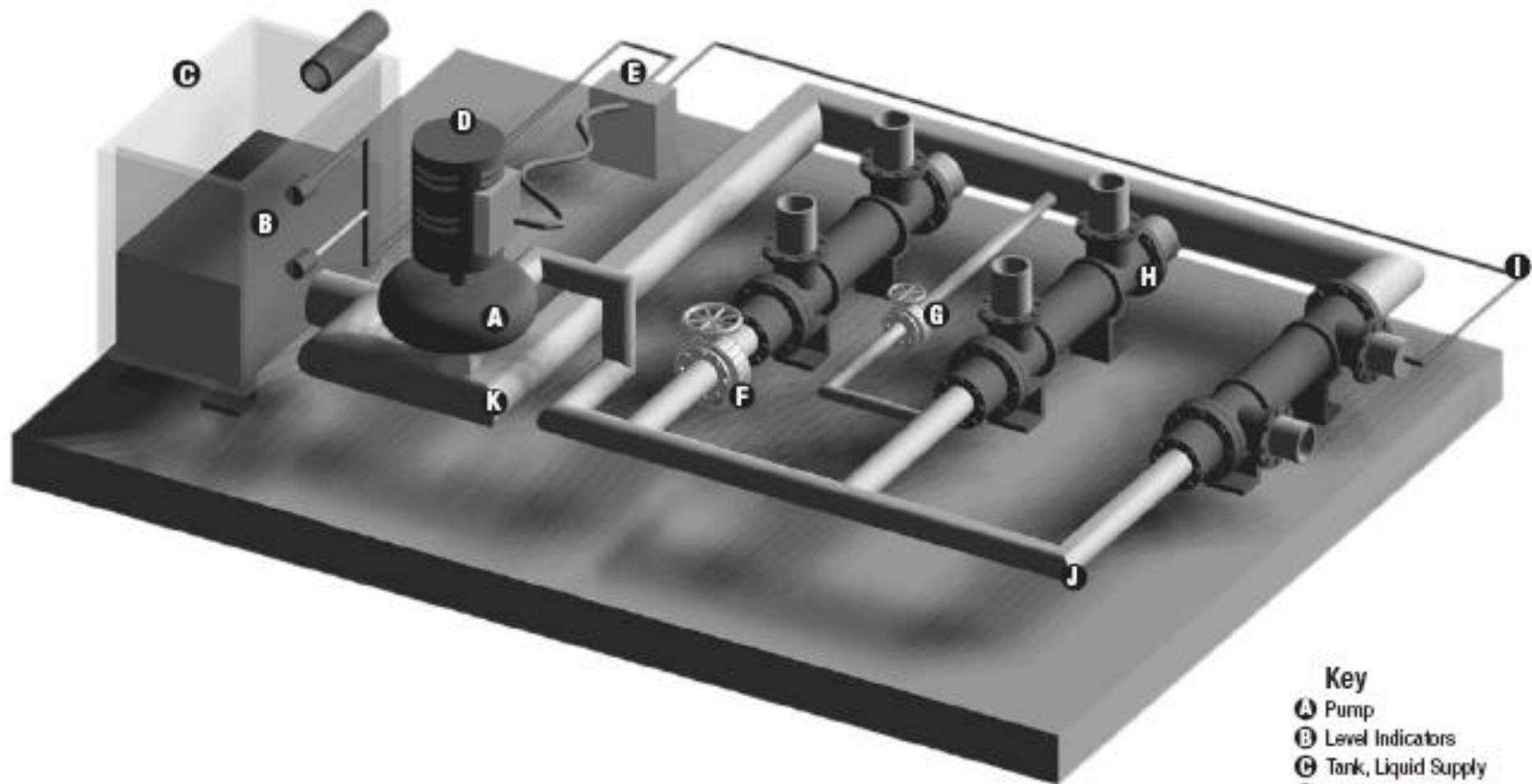
# **Objective of Pumping System**

- **Transfer liquid from source to destination**
- **Circulate liquid around a system**



# What Are Pumping Systems

- **Main pump components**
  - **Pumps**
  - **Prime movers: electric motors, diesel engines, air system**
  - **Piping to carry fluid**
  - **Valves to control flow in system**
  - **Other fittings, control, instrumentation**
- **End-use equipment**
  - **Heat exchangers, tanks, hydraulic machines**



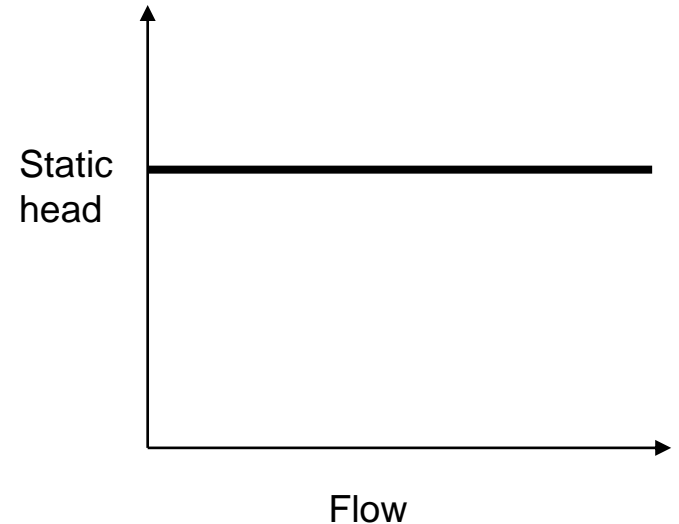
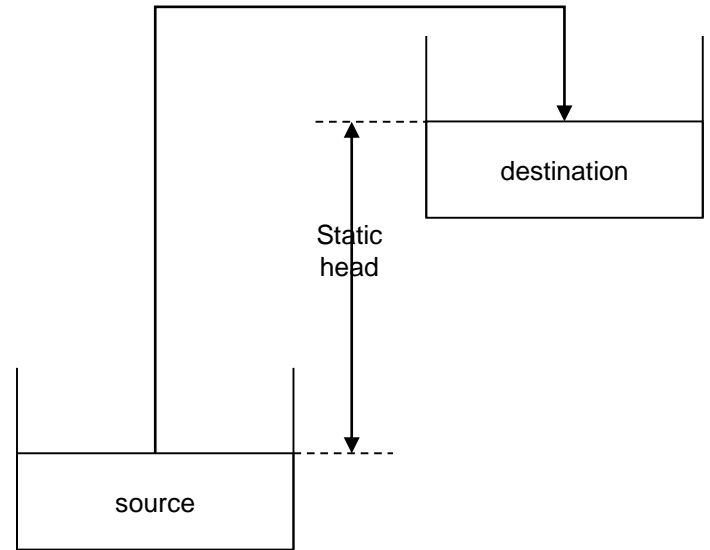
### Key

- Ⓐ Pump
- Ⓑ Level Indicators
- Ⓒ Tank, Liquid Supply
- Ⓓ Pump Motor
- Ⓔ Motor Controller
- Ⓕ Throttle Valve
- Ⓖ Bypass Valve
- Ⓗ Heat Exchangers  
*(End-Use Equipment)*
- Ⓛ Instrumentation Line
- Ⓜ Pump Discharge Piping
- Ⓝ Pump Suction Piping

Figure 1. Typical Pumping System Components

# Pumping Head

- **Head**
  - Resistance of the system
  - Two types: static and friction
- **Static head**
  - Difference in height between source and destination
  - Independent of flow





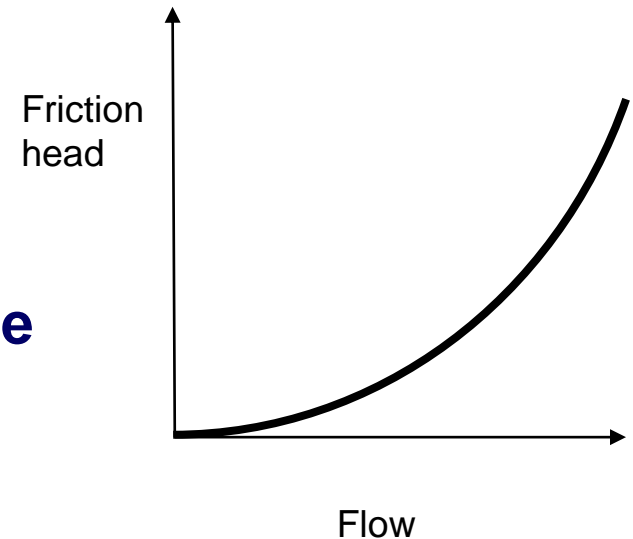
# Pumping System Characteristics

- **Static head consists of**
  - **Static suction head (h<sub>S</sub>):** lifting liquid relative to pump center line
  - **Static discharge head (h<sub>D</sub>)** vertical distance between centerline and liquid surface in destination tank
- **Static head at certain pressure**

$$\text{Head (in feet)} = \frac{\text{Pressure (psi)} \times 2.31}{\text{Specific gravity}}$$

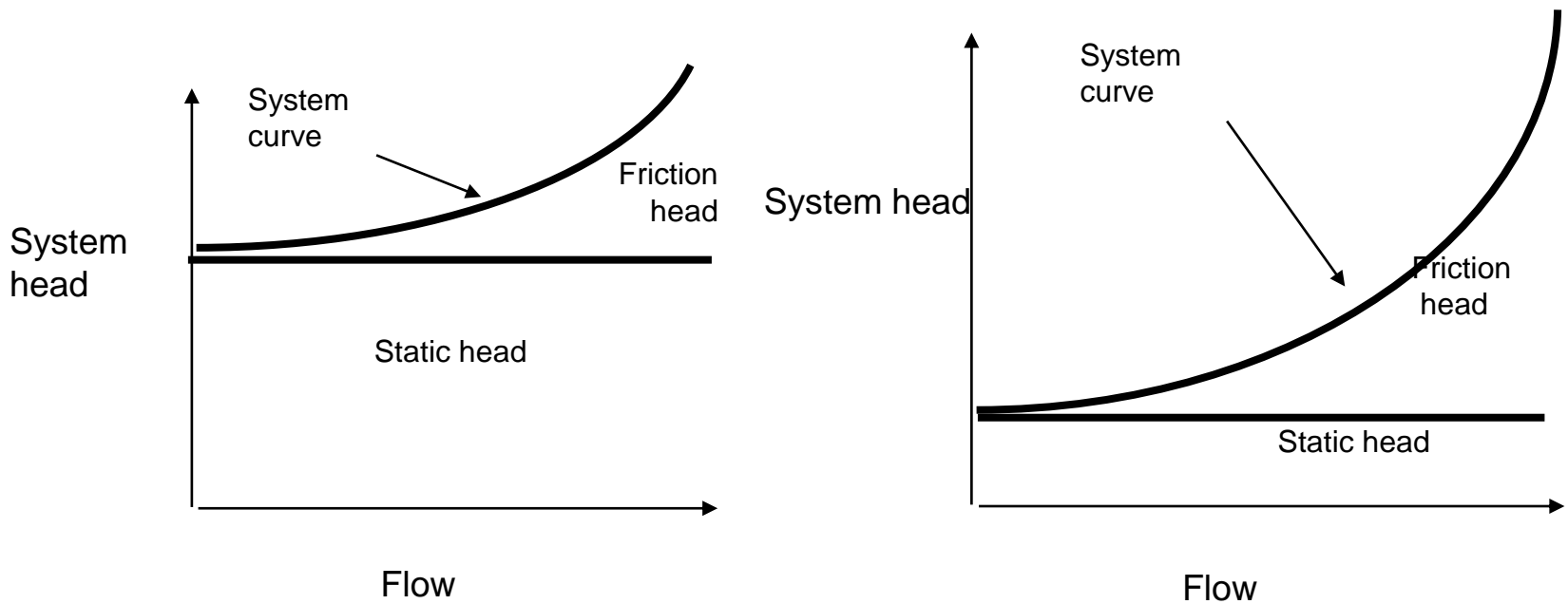
# Friction Head

- **Resistance to flow in pipe and fittings**
- **Depends on size, pipes, pipe fittings, flow rate, nature of liquid**
- **Proportional to square of flow rate**
- **Closed loop system only has friction head (no static head)**



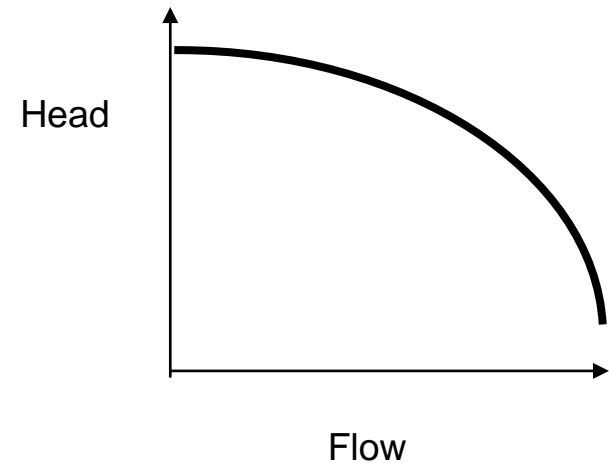
**In most cases:**

**Total head = Static head + friction head**



# Pump Performance Curve

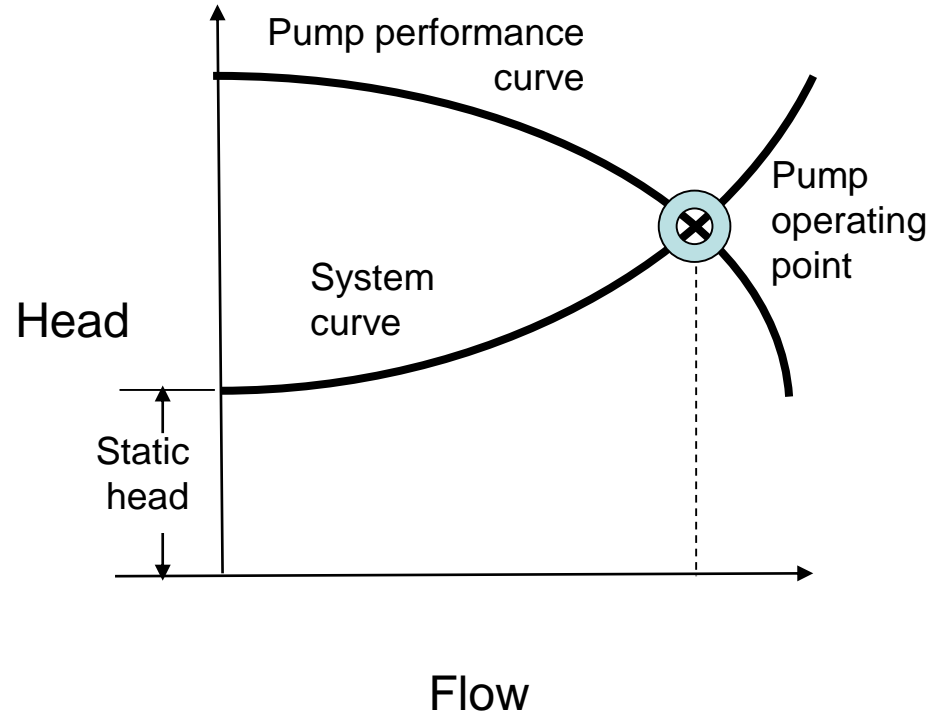
- **Relationship between head and flow**
  - **Flow increase**
  - **System resistance increases**
  - **Head increases**
  - **Flow decreases to zero**
- **Zero flow rate: risk of pump burnout**



Performance curve for centrifugal pump

# Pump Operating Point

- **Duty point: rate of flow at certain head**
- **Pump operating point: intersection of pump curve and system curve**



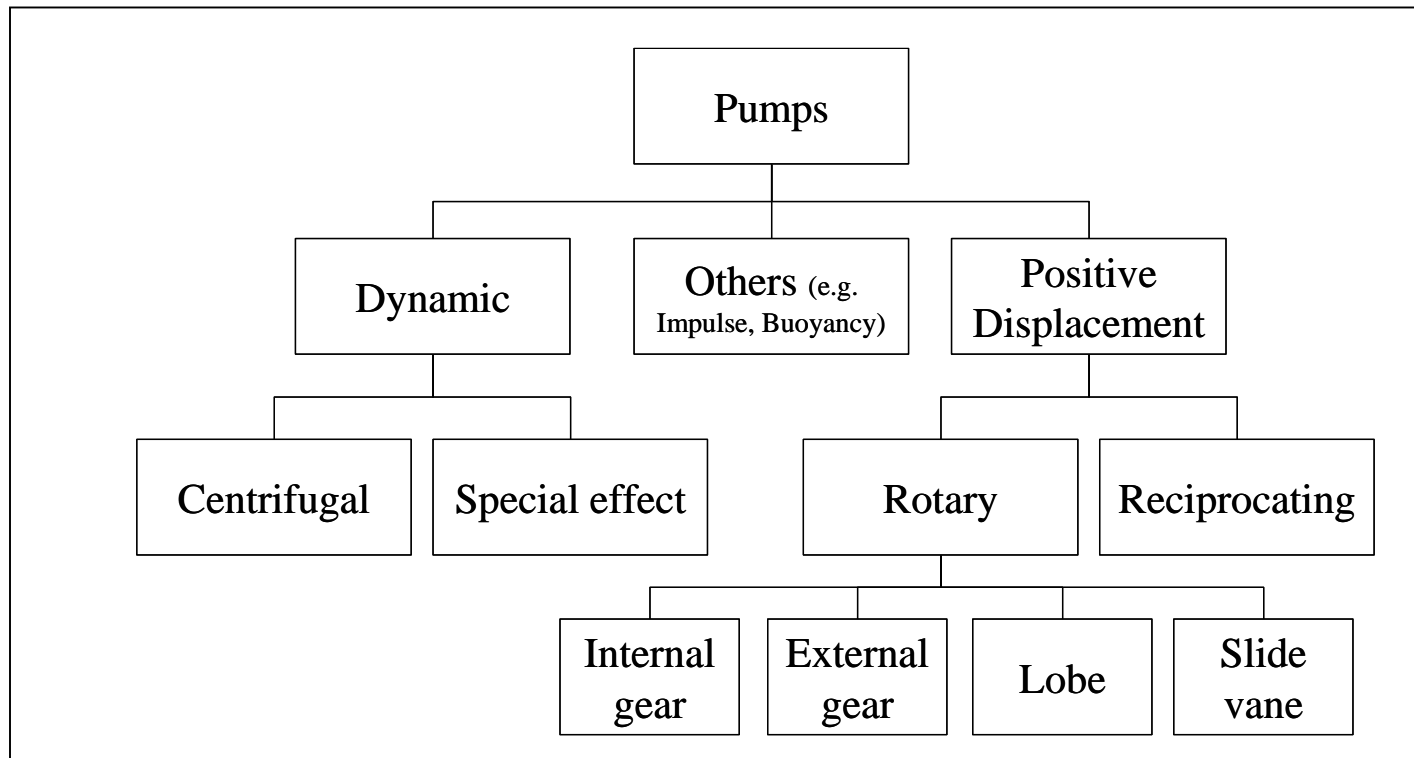
# **Pump Suction Performance (NPSH)**

- **Cavitation or vaporization: bubbles inside pump**
- **If vapor bubbles collapse**
  - **Erosion of vane surfaces**
  - **Increased noise and vibration**
  - **Choking of impeller passages**
- **Net Positive Suction Head**
  - **NPSH Available: how much pump suction exceeds liquid vapor pressure**
  - **NPSH Required: pump suction needed to avoid cavitation**

# Type of Pumps

## Pump Classification

Classified by operating principle



# Positive Displacement Pumps

- **For each pump revolution**
  - Fixed amount of liquid taken from one end
  - Positively discharged at other end
- **If pipe blocked**
  - Pressure rises
  - Can damage pump
- **Used for pumping fluids other than water**



# Positive Displacement Pumps

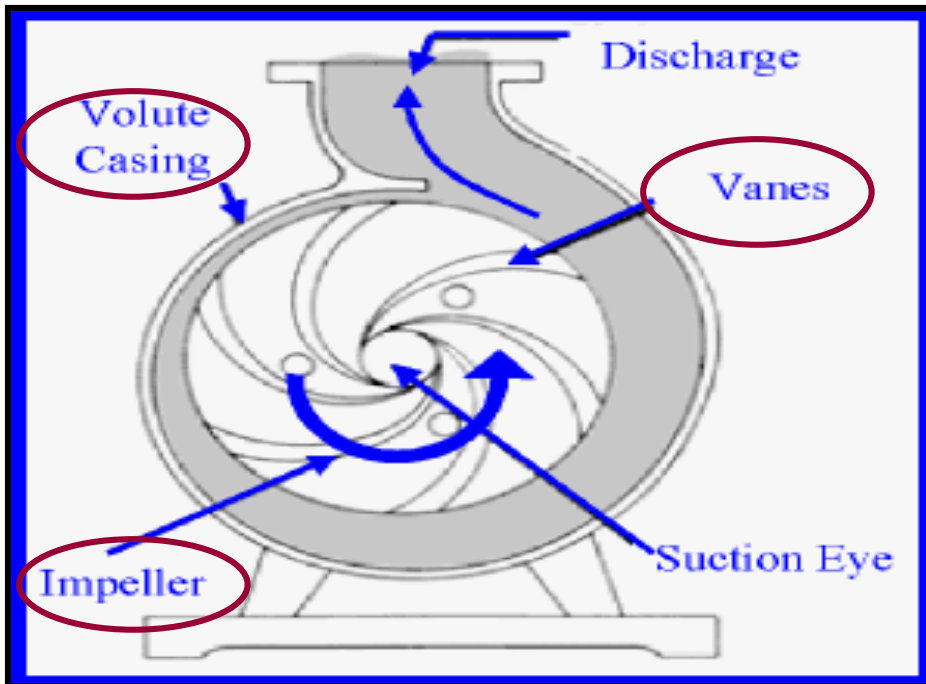
- **Reciprocating pump**
  - Displacement by reciprocation of piston plunger
  - Used only for viscous fluids and oil wells
- **Rotary pump**
  - Displacement by rotary action of gear, cam or vanes
  - Several sub-types
  - Used for special services in industry

# Dynamic Pumps

- **Mode of operation**
  - Rotating impeller converts kinetic energy into pressure or velocity to pump the fluid
- **Two types**
  - Centrifugal pumps: pumping water in industry – 75% of pumps installed
  - Special effect pumps: specialized conditions

# Centrifugal Pumps

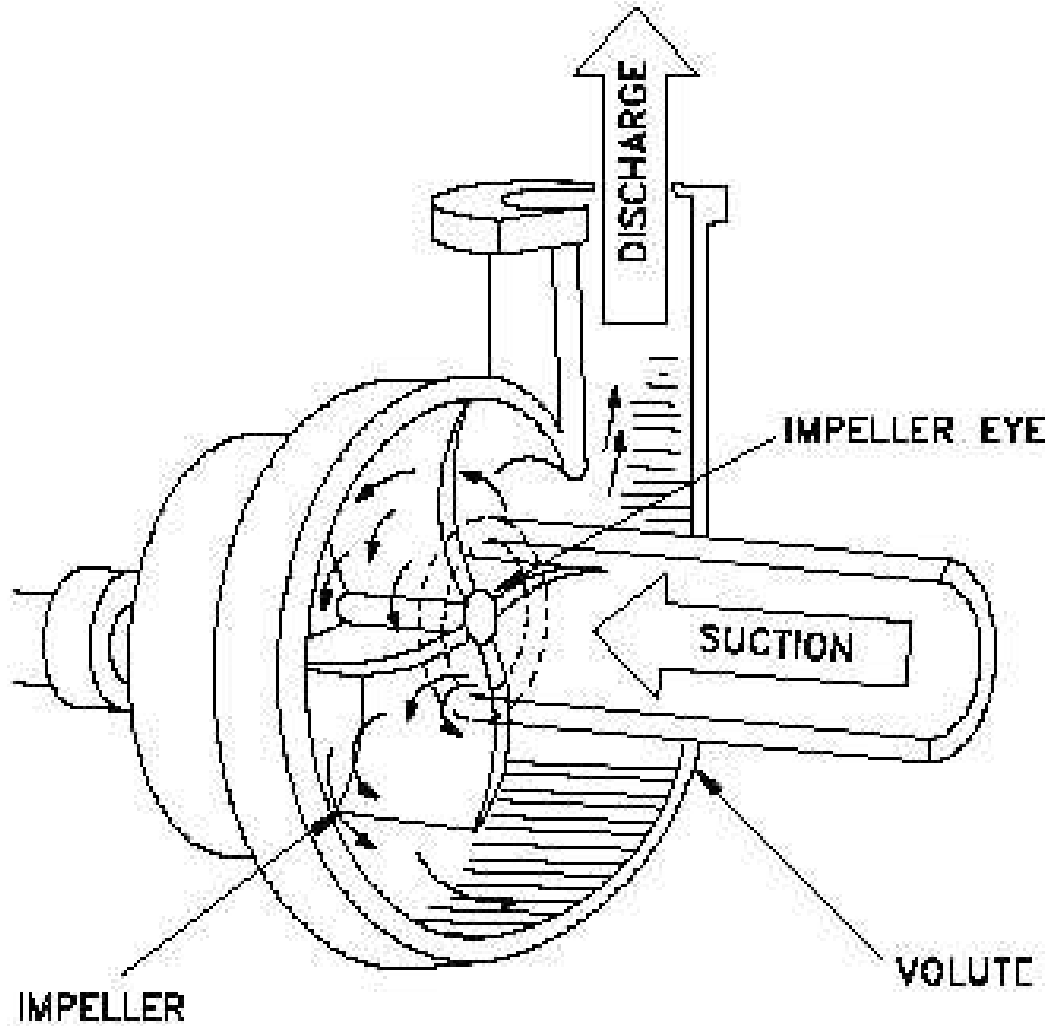
## How do they work?



- Liquid forced into impeller
- Vanes pass kinetic energy to liquid: liquid rotates and leaves impeller
- Volute casing converts kinetic energy into pressure energy

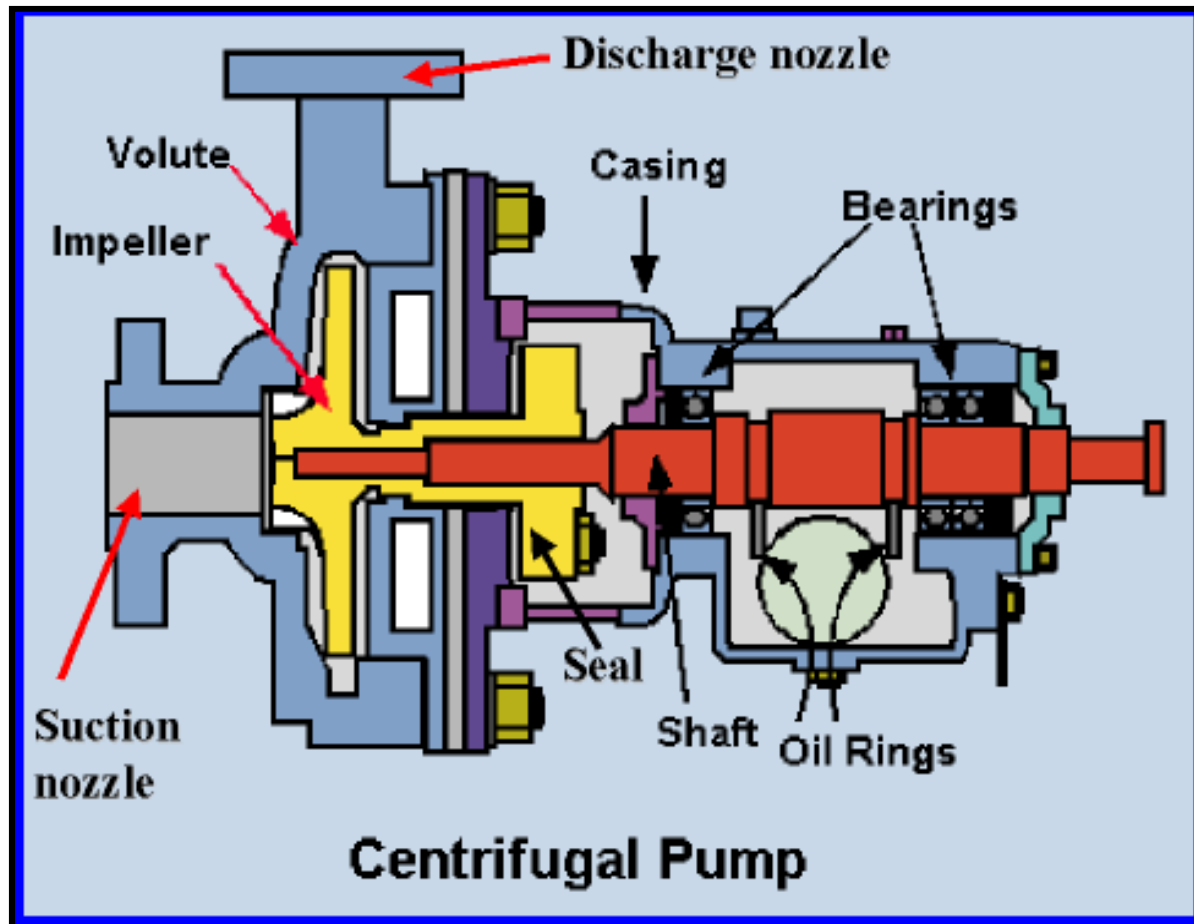
# CENTRIFUGAL PUMP

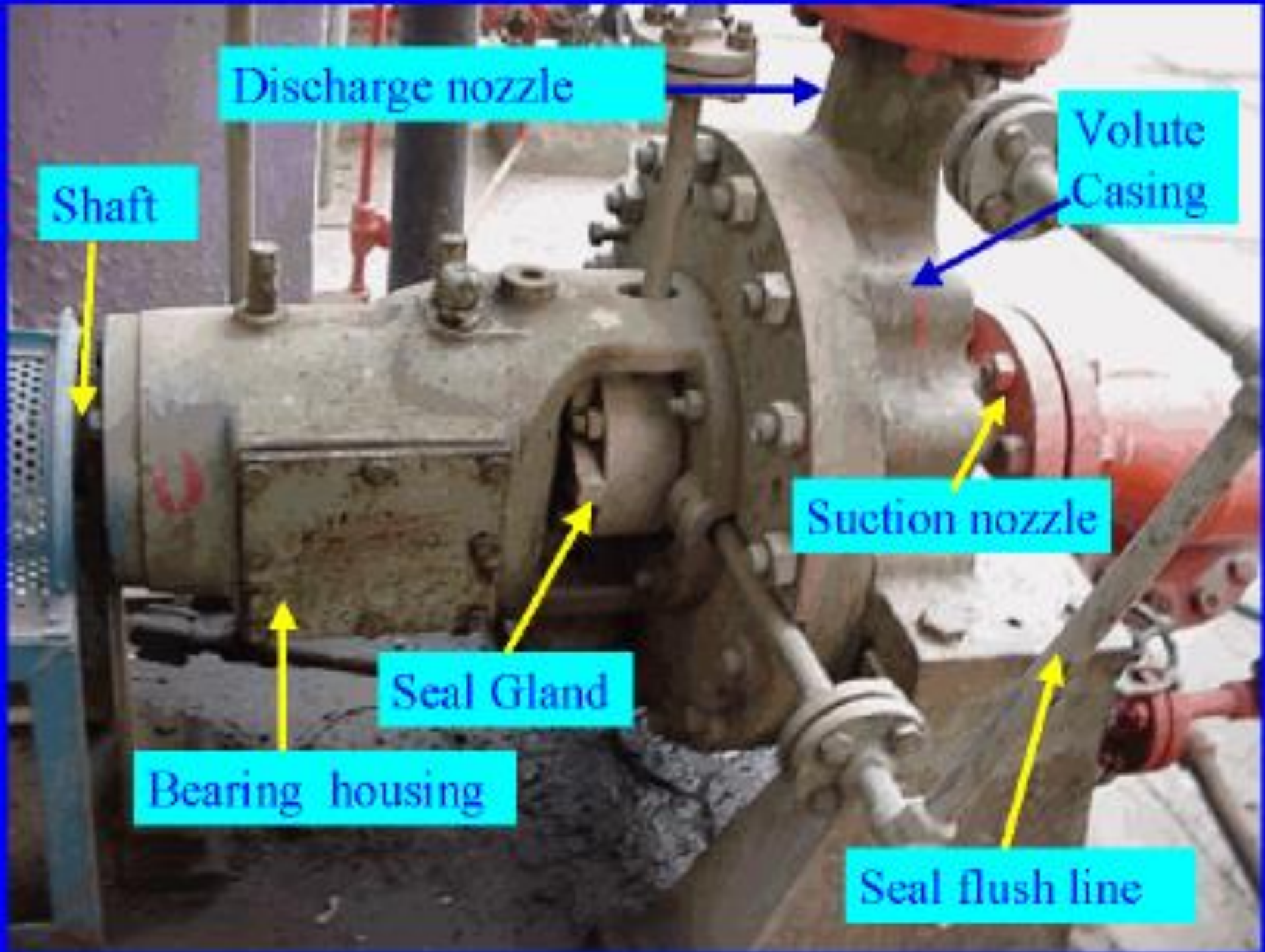
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# Centrifugal Pumps

## Rotating and stationary components

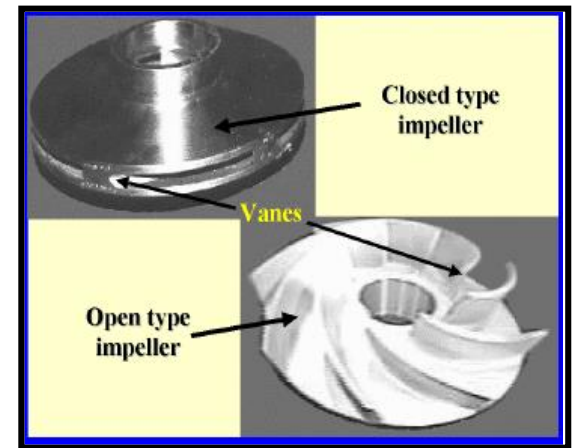




# Centrifugal Pumps

## Impeller

- Main rotating part that provides centrifugal acceleration to the fluid
- Number of impellers = number of pump stages
- Impeller classification: direction of flow, suction type and shape/mechanical construction



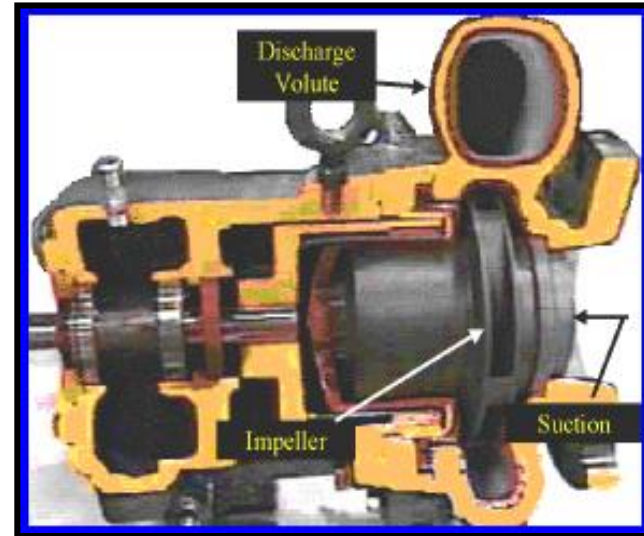
## Shaft

- Transfers torque from motor to impeller during pump start up and operation

# Centrifugal Pumps

## Casings

- **Functions**
  - Enclose impeller as “pressure vessel”
  - Support and bearing for shaft and impeller
- **Volute case**
  - Impellers inside casings
  - Balances hydraulic pressure on pump shaft
- **Circular casing**
  - Vanes surrounds impeller
  - Used for multi-stage pumps





# PUMP CALCULATIONS

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$$\text{Pump Efficiency} = \frac{\text{Hydrolic power, } P_h \times 100}{\text{Power input to the pump shaft}}$$

Where,

$$\text{Hydraulic power } P_h (\text{kW}) = Q (\text{m}^3/\text{s}) \times \text{Total head, } (h_d - h_s) (\text{m}) \times \rho (\text{kg}/\text{m}^3) \times g (\text{m}/\text{s}^2) / 1000$$

Q=Volume flow rate,  $\rho$ =density of the fluid,

g=acceleration due to gravity

$h_d$  = Delivery head,  $h_s$  = Suctionhead

# POWER CALCULATIONS

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Assume that we need to pump 68 m<sup>3</sup>/hr to a 47 meter head with a pump that is 60% efficient at that point, motor efficiency 90%.

**Calculate motor power.**

$$\begin{aligned}\text{Liquid Power} &= 68 * 47 * 1000 * 9.81 / 3600 * 1000 \\ &= 8.7 \text{ kW}\end{aligned}$$

$$\text{Shaft Power} = 8.7 / 0.60 = 14.5 \text{ kW}$$

$$\text{Motor Power} = 14.5 / 0.9 = 16.1 \text{ kW}$$

# Pump Efficiency Example

Illustration of calculation method outlined

A chemical plant operates a cooling water pump for process cooling and refrigeration applications. During the performance testing the following operating parameters were measured;

## Measured Data

Pump flow, $Q$	0.40 m <sup>3</sup> / s
Power absorbed, $P$	325 kW
Suction head (Tower basin level), $h_1$	+1 M
Delivery head, $h_2$	55 M
Height of cooling tower	5 M
Motor efficiency	88 %
Type of drive	Direct coupled
Density of water	996 kg/ m <sup>3</sup>

# Pump Efficiency Example

Flow delivered by the pump	:	0.40 m <sup>3</sup> /s
Total head, h <sub>2</sub> -(+h <sub>1</sub> )	:	54 M
Hydraulic power	:	0.40 x 54 x 996 x 9.81/1000 = 211 kW
Actual power consumption	:	325 kW
Overall system efficiency	:	(211 x 100) / 325 = 65 %
Pump efficiency	:	65/0.88 = 74 %

# Assessment of Pumps

## How to Calculate Pump Performance

- **Pump shaft power ( $P_s$ ) is actual horsepower delivered to the pump shaft**

*Pump shaft power ( $P_s$ ):*

$$P_s = \text{Hydraulic power } H_p / \text{pump efficiency } \eta_{\text{Pump}}$$

*Pump Efficiency ( $\eta_{\text{Pump}}$ ):*

$$\eta_{\text{Pump}} = \text{Hydraulic Power} / \text{Pump Shaft Power}$$

- **Pump output/Hydraulic/Water horsepower ( $H_p$ ) is the liquid horsepower delivered by the pump**

*Hydraulic power ( $H_p$ ):*

$$H_p = Q \text{ (m}^3\text{/s)} \times \text{Total head, } h_d - h_s \text{ (m)} \times \rho \text{ (kg/m}^3\text{)} \times g \text{ (m/s}^2\text{)} / 1000$$

$h_d$  - discharge head  
 $\rho$  - density of the fluid

$h_s$  – suction head,  
 $g$  – acceleration due to gravity

# **Difficulties in Pump Assessment**

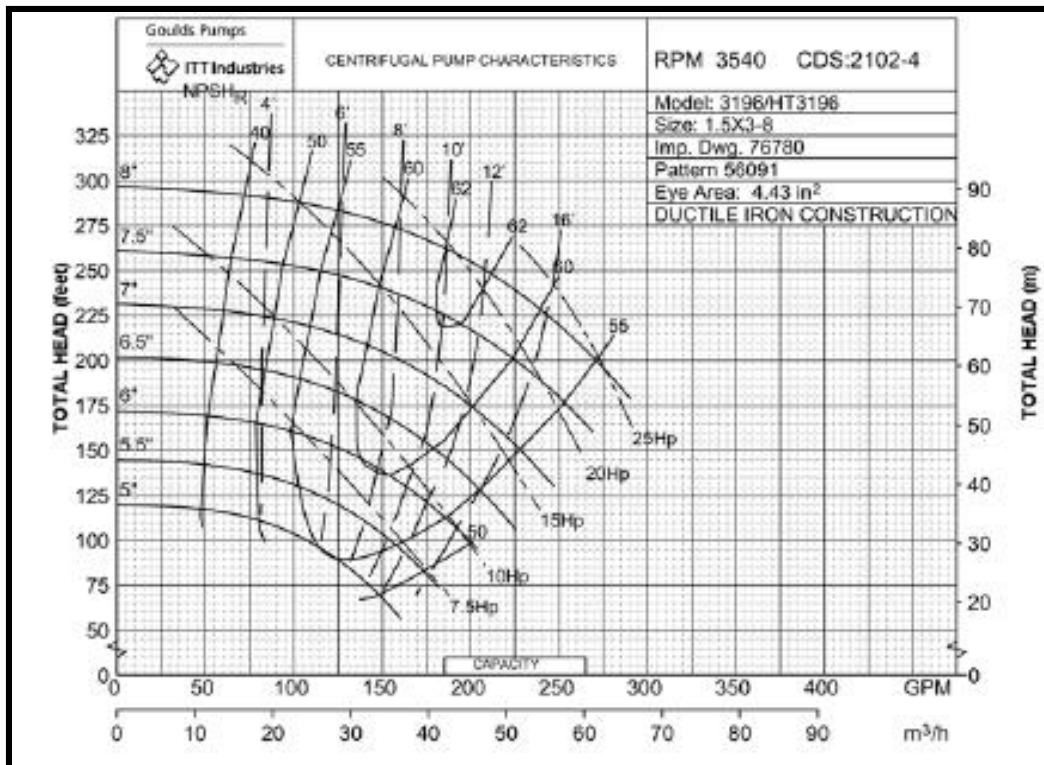
- **Absence of pump specification data to assess pump performance**
- **Difficulties in flow measurement and flows are often estimated**
- **Improper calibration of pressure gauges & measuring instruments**
  - **Calibration not always carried out**
  - **Correction factors used**

# **Energy Efficiency Opportunities**

- 1. Selecting the right pump**
- 2. Controlling the flow rate by speed variation**
- 3. Pumps in parallel to meet varying demand**
- 4. Eliminating flow control valve**
- 5. Eliminating by-pass control**
- 6. Start/stop control of pump**
- 7. Impeller trimming**

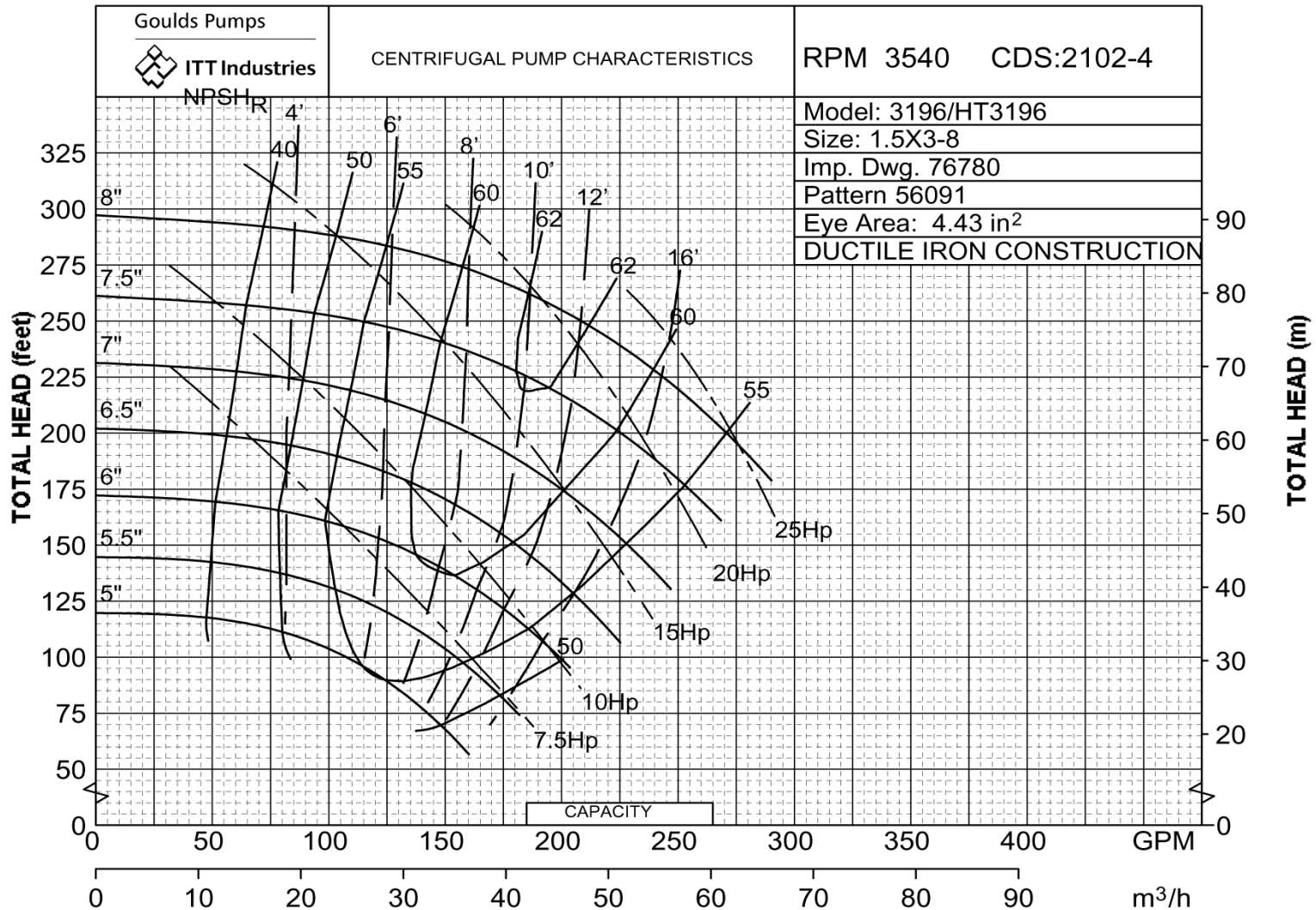
# 1. Selecting the Right Pump

## Pump performance curve for centrifugal pump





# TYPICAL PUMP CHARACTERISTIC CURVES



- **Oversized pump**
  - Requires flow control (throttle valve or by-pass line)
  - Provides additional head
  - System curve shifts to left
  - Pump efficiency is reduced
- **Solutions if pump already purchased**
  - VSDs or two-speed drives
  - Lower RPM
  - Smaller or trimmed impeller

## 2. Controlling Flow: speed variation

### Explaining the effect of speed

- **Affinity laws: relation speed  $N$  and**
  - Flow rate  $Q \propto N$
  - Head  $H \propto N^2$
  - Power  $P \propto N^3$
- **Small speed reduction (e.g.  $\frac{1}{2}$ ) = large power reduction (e.g.  $\frac{1}{8}$ )**

## **2. Controlling Flow: speed variation**

### **Variable Speed Drives (VSD)**

- **Speed adjustment over continuous range**
- **Power consumption also reduced!**
- **Two types**
  - **Mechanical: hydraulic clutches, fluid couplings, adjustable belts and pulleys**
  - **Electrical: eddy current clutches, wound-rotor motor controllers, Variable Frequency Drives (VFDs)**

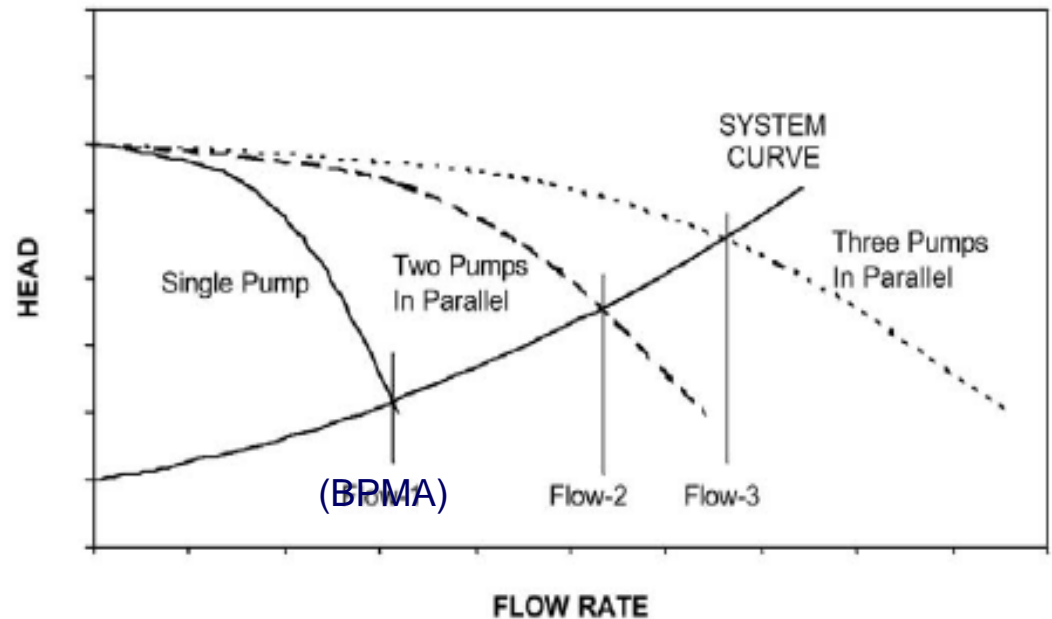
## 2. Controlling Flow: speed variation

### Benefits of VSDs

- **Energy savings** (*not just reduced flow!*)
- **Improved process control**
- **Improved system reliability**
- **Reduced capital and maintenance costs**
- **Soft starter capability**

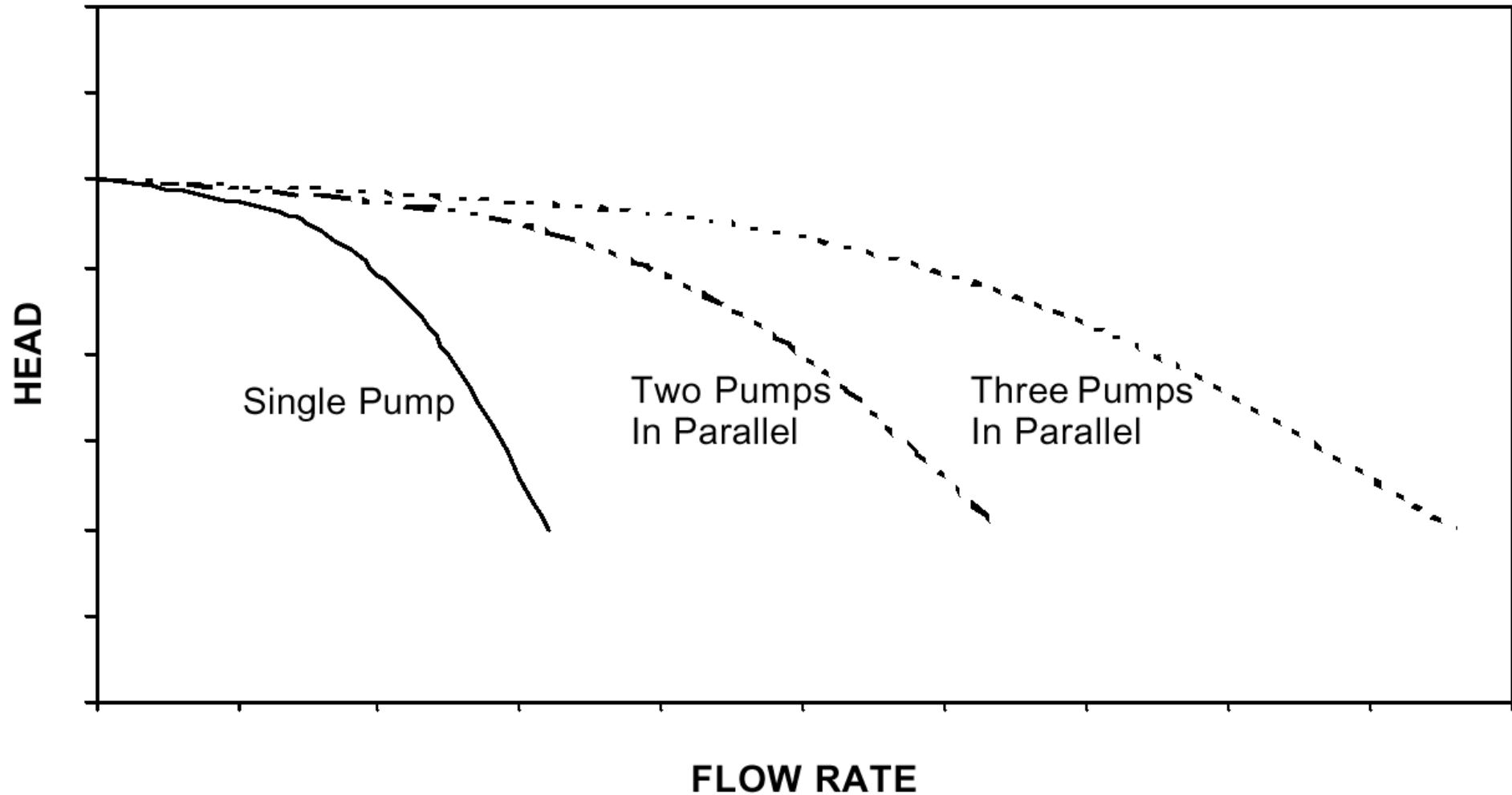
# 3. Parallel Pumps for Varying Demand

- Multiple pumps: some turned off during low demand
- Used when static head is  $> 50\%$  of total head
- System curve does not change
- Flow rate lower than sum of individual flow rates



# PUMPS IN PARALLEL OPERATION

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# CENTRIFUGAL PUMPS IN PARALLEL

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- The total head for the **combination** is the same as the total head for each pump

$$\Delta h_T = \Delta h_1 = \Delta h_2$$

- The flowrate or capacity is the sum of the two pumps

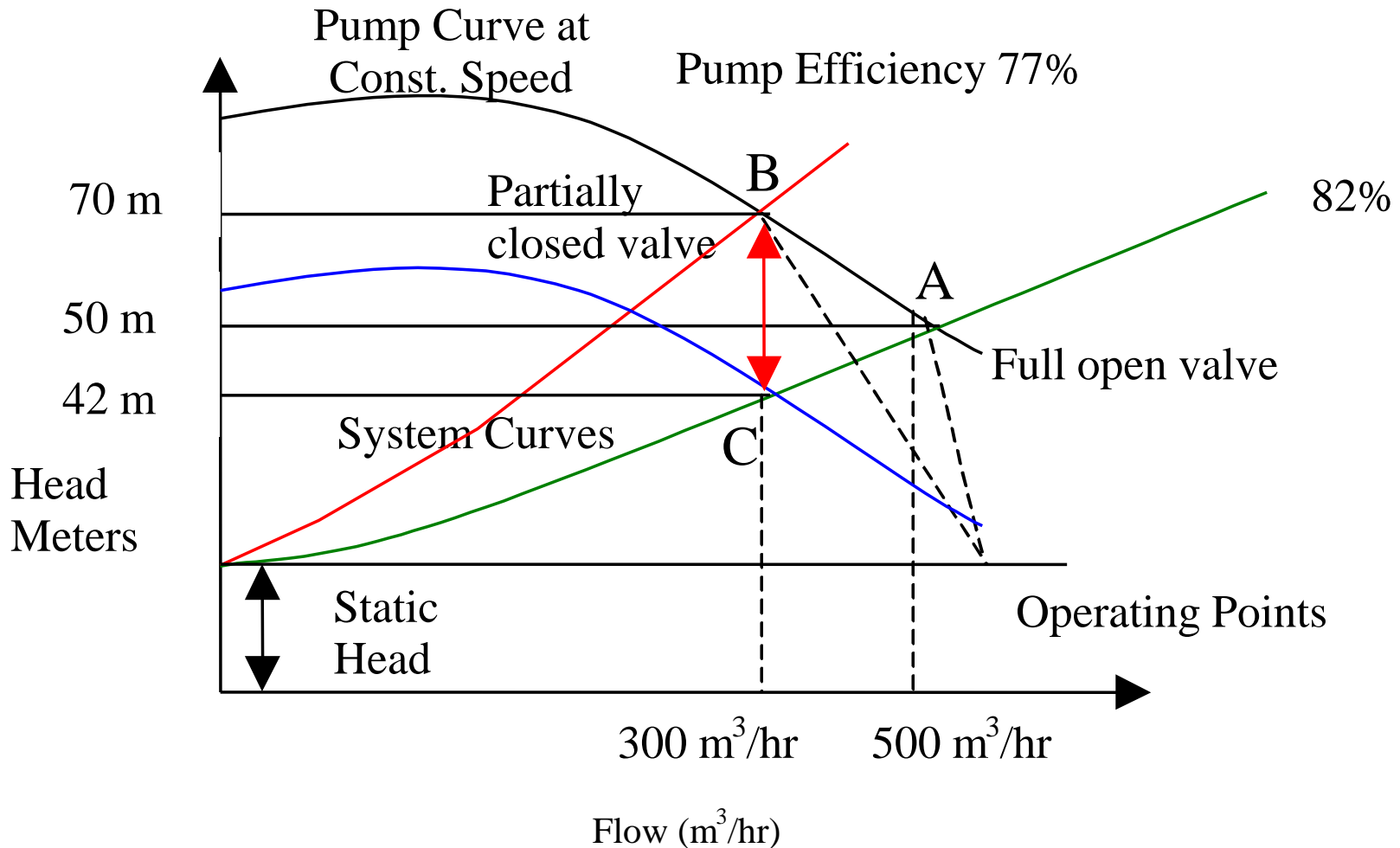
$$Q_T = Q_1 + Q_2$$



## **4. Eliminating By-pass Control**

- **Pump discharge divided into two flows**
  - **One pipeline delivers fluid to destination**
  - **Second pipeline returns fluid to the source**
- **Energy wastage because part of fluid pumped around for no reason**

# EFFECT OF THROTTLING



## **5. Eliminating By-pass Control**

- **Pump discharge divided into two flows**
  - **One pipeline delivers fluid to destination**
  - **Second pipeline returns fluid to the source**
- **Energy wastage because part of fluid pumped around for no reason**

## **6. Start/Stop Control of Pump**

- **Stop the pump when not needed**
- **Example:**
  - **Filling of storage tank**
  - **Controllers in tank to start/stop**
- **Suitable if not done too frequently**
- **Method to lower the maximum demand (pumping at non-peak hours)**

# 7. Impeller Trimming

- **Changing diameter: change in velocity**
- **Considerations**
  - **Cannot be used with varying flows**
  - **No trimming >25% of impeller size**
  - **Impeller trimming same on all sides**
  - **Changing impeller is better option but more expensive and not always possible**

# THE AFFINITY LAW FOR A CENTRIFUGAL PUMP

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**Flow:**

$$Q1 / Q2 = N1 / N2$$

Example:

$$100 / Q2 = 1750/3500$$

$$Q2 = 200 \text{ m}^3/\text{hr}$$

**Head:**

$$H1/H2 = (N1^2) / (N2^2)$$

Example:

$$100 / H2 = 1750^2 / 3500^2$$

$$H2 = 400 \text{ m}$$

**Power :**

$$P1 / P2 = (N1^3) / (N2^3)$$

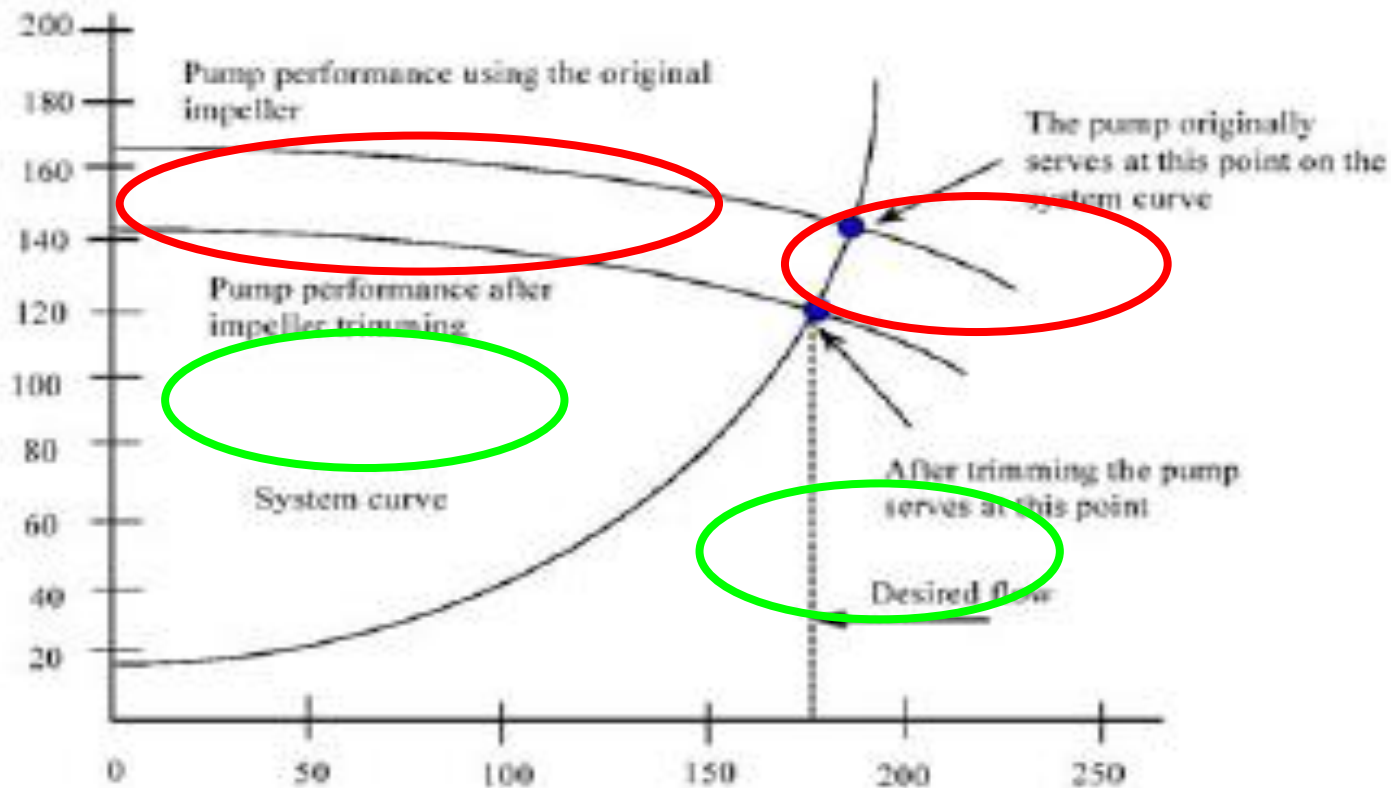
Example:

$$5/P2 = 1750^3 / 3500^3$$

$$P2 = 40$$

# 7. Impeller Trimming

## Impeller trimming and centrifugal pump performance



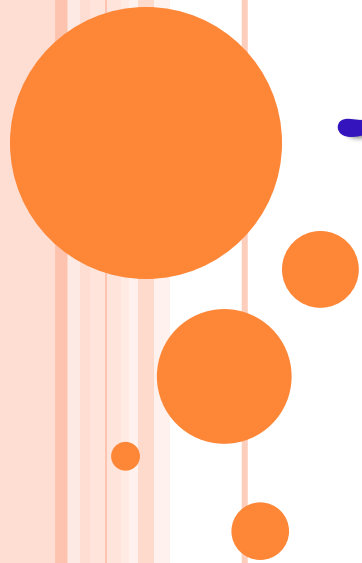
# Comparing Energy Efficiency Options

<b>Parameter</b>	<b>Change control valve</b>	<b>Trim impeller</b>	<b>VFD</b>
<b>Impeller diameter</b>	430 mm	375 mm	430 mm
<b>Pump head</b>	71.7 m	42 m	34.5 m
<b>Pump efficiency</b>	75.1%	72.1%	77%
<b>Rate of flow</b>	80 m <sup>3</sup> /hr	80 m <sup>3</sup> /hr	80 m <sup>3</sup> /hr
<b>Power consumed</b>	23.1 kW	14 kW	11.6 kW



*THANK YOU*

*for your attention*



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